# Introduction to the Kinematics of Rigid Bodies

#### François Faure

Grenoble Université

< □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □ > < □

### **Motivation**

- Given the desired displacement of a point
- how to compute the necessary joint motions ?



# A moving frame

- ► Frame R<sub>1</sub> is moving wrt. reference frame R<sub>0</sub>
- Vector  $\mathbf{u} = O_1 P$  is fixed in  $\mathcal{R}_1$
- We write  ${}^{0}\mathbf{u}$  its coordinates in  $\mathcal{R}_{0}$
- We write <sup>0</sup> u the derivative of <sup>0</sup>u
- Let **R**(dt) the rotation of  $\mathcal{R}_1$  from time *t* to t + dt.

$$\mathbf{u}(t + dt) = \mathbf{R}(dt)\mathbf{u}(t)$$
  
$$\mathbf{u}(t + dt) - \mathbf{u}(t) = (\mathbf{R}(dt) - \mathbf{I})\mathbf{u}(t)$$

where I is the identity matrix.

	$\leq$	X	Р /	
Ć		01	Ŷ	~
	0			·
-	· • · · · ·	▶ ▲ ≧ ▶	<.≣> ≣	୬ ୦ 3/:

#### Angular velocity vector

• Let 
$$\mathbf{n} = \begin{bmatrix} n_x \\ n_y \\ n_z \end{bmatrix}$$
 and  $d\theta = \dot{\theta} dt$  be the axis and angle of rotation of  $\mathbf{R}(dt)$ 

R(dt)-I is close to

$$\begin{bmatrix} 0 & -n_z \dot{\theta} dt & n_y \dot{\theta} dt \\ n_z \dot{\theta} dt & 0 & -n_x \dot{\theta} dt \\ -n_y \dot{\theta} dt & n_x \dot{\theta} dt & 0 \end{bmatrix} = \dot{\theta} dt [n \times ]$$

where matrix  $[n \times] = \begin{bmatrix} 0 & -n_z & n_y \\ n_z & 0 & -n_x \\ -n_y & n_x & 0 \end{bmatrix}$ is the cross product matrix:  $[n \times] \mathbf{u} = \mathbf{n} \times \mathbf{u}$ 

We call Ω<sub>1/0</sub> = θn the angular velocity vector of frame R<sub>1</sub> wrt. frame R<sub>0</sub>

# Derivative of a constant vector in a moving frame

► For **u=0**<sub>1</sub>**P** constant in frame *R*<sub>1</sub>:

$$\hat{\boldsymbol{\mathsf{U}}} = \hat{\boldsymbol{\mathsf{R}}} \boldsymbol{\mathsf{u}}$$
  
=  $[\Omega_{1/0} \times] \boldsymbol{\mathsf{u}}$   
=  $\Omega_{1/0} \times \boldsymbol{\mathsf{u}}$ 

- <sup>0</sup> u can be expressed in any reference frame
- ► the translation of R<sub>1</sub> wrt. R<sub>0</sub> has no influence on u



### Velocity of a point attached to a moving frame



#### Acceleration of a point attached to a moving frame

- Deriving the velocity equation
- and noticing that  $\overrightarrow{O_1P}$  is fixed in  $\mathcal{R}_1$ , we get

$$\Gamma_{\mathcal{A}}^{1/0} = \Gamma_{\mathcal{O}_{1}}^{1/0} + \dot{\Omega}_{1/0} \times \overrightarrow{\mathcal{O}_{1}\mathcal{A}} + \Omega_{1/0} \times \left(\Omega_{1/0} \times \overrightarrow{\mathcal{O}_{1}\mathcal{A}}\right)$$

- $\Gamma_A^{1/0}$  is the linear acceleration of the origin
- $\dot{\Omega}_{1/0} \times \overrightarrow{O_1 A}$  encodes the angular acceleration
- $\Omega_{1/0} \times \left( \Omega_{1/0} \times \overrightarrow{O_1 A} \right)$  is the centripetal acceleration due to the rotation velocity

### Derivative of a vector moving in a moving frame

• Let 
$$(\mathbf{e}_1, \mathbf{e}_e, \mathbf{e}_3)$$
 be a basis of  $\mathcal{R}_1$ 

We thus write

$${}^{1}\mathbf{u} = \sum_{i} x_{i}\mathbf{e}_{i}$$
$$\dot{\mathbf{u}} = \sum_{i} \dot{x}_{i}\mathbf{e}_{i} + \sum_{i} x_{i}\dot{\mathbf{e}}_{i}$$

hence

$${}^{0}\dot{\boldsymbol{u}}={}^{1}\dot{\boldsymbol{u}}+\boldsymbol{\Omega}_{1/0}\times\boldsymbol{u}$$

#### Velocity of a point moving in a moving frame

- Let  $V_A^{/1}$  be the velocity of point A wrt.  $\mathcal{R}_1$
- We add it to the velocity in R<sub>0</sub> of a point at the same place and fixed in R<sub>1</sub>:

$$V_A^{/0} = V_A^{/1} + V_{O_1}^{1/0} + \Omega_{1/0} \times \overrightarrow{O_1 A}$$

#### Acceleration of a point moving in a moving frame

By differentiating the velocity, we get:

$$\Gamma_{A}^{/0} = \underbrace{\Gamma_{A}^{/1} + \Omega_{1/0} \times V_{A}^{/1}}_{\dot{V}_{A}^{/1}} + \Gamma_{O_{1}}^{/0} + \underbrace{\dot{\Omega}_{1/0} \times \mathbf{O}_{1}A + \Omega_{1/0} \times V_{A}^{/1} + \Omega_{1/0} \times (\Omega_{1/0} \times \overrightarrow{O_{1}A})}_{\Omega_{1/0} \times \overrightarrow{O_{1}A}}$$

# Acceleration of a point moving in a moving frame (continued)

and then:

$$\Gamma_{A}^{/0} = \Gamma_{A}^{/1} + \Gamma_{O_{1}}^{/0} + \Omega_{1/0} \times (\Omega_{1/0} \times \overrightarrow{O_{1}A}) + 2\Omega_{1/0} \times V_{A}^{/1}$$

- with
  - $\Gamma_A^{/1} = \sum_i \ddot{x}_i \mathbf{e}_i$  relative acceleration
  - $\Gamma_{O_1}^{/0}$  linear acceleration of the moving frame
  - $\Omega_{1/0} \times (\Omega_{1/0} \times \overrightarrow{O_1 A})$  centripetal acceleration
  - $2\Omega_{1/0} \times V_A^{/1}$  Coriolis acceleration

#### Velocity of articulated bodies

The recursive use of the velocity equation gives:

$$V_{A}^{2/0} = V_{A}^{2/1} + V_{O_{1}}^{1/0} + \Omega_{1/0} \times \overrightarrow{O_{1}A}$$
$$= V_{A}^{2/1} + V_{A}^{1/0}$$

and more generally

$$V_A^{n/0} = \sum_{i=1}^n V_A^{i/i-1}$$



#### **Joints**

Defined by the allowed relative motions



### **More Joints**



14/35

#### Joint transforms

 Generally, the transform between two articulated bodies can be written as a product of three transforms

$${}^{i-1}_{i}\mathbf{C} = ({}^{i-1}_{i}\mathbf{C}_{\mathbf{p}})({}^{i-1}_{i}\mathbf{C}_{\mathbf{l}})({}^{i-1}_{i}\mathbf{C}_{\mathbf{c}})$$



#### The Denavit-Hartenberg model

One axis per joint, with one translation and one rotation



Recursive transform computation in the Denavit-Hartenberg model

Recursive velocity computation in the Denavit-Hartenberg model

 $\overrightarrow{OA} = {}^{n}\overrightarrow{O_{n}A}$   $\overrightarrow{V} = \overrightarrow{0}$   $\Omega = \overrightarrow{0}$ for i in n..1  ${}^{i-1}_{i}\mathbf{C} = \mathbf{T}_{\mathbf{x},\mathbf{a}_{i-1}}\mathbf{R}_{\mathbf{x},\alpha_{i-1}}\mathbf{T}_{\mathbf{z},\mathbf{d}_{i}}\mathbf{R}_{\mathbf{z},\theta_{i}}$   $\Omega = {}^{i-1}_{i}\mathbf{B}(\Omega + \dot{\theta}_{i}\mathbf{z})$   $\overrightarrow{V} = {}^{i-1}_{i}\mathbf{B}(\overrightarrow{V} + \dot{d}_{i}\mathbf{z} + \dot{\theta}_{i}\mathbf{z} \times \overrightarrow{OA})$   $\overrightarrow{OA} = {}^{i-1}_{i}\mathbf{C}\overrightarrow{OA}$ 

#### Inverse kinematics

- Given the desired displacement of a point
- how to compute the necessary joint motions ?



#### Linear equations

- Translational joints
- Point and target



matrix equation:

$$\begin{pmatrix} a_{1x} & a_{2x} \\ a_{1y} & a_{2y} \end{pmatrix} \begin{pmatrix} \Delta q_1 \\ \Delta q_2 \end{pmatrix} = \begin{pmatrix} c_x \\ c_y \end{pmatrix}$$

# A single scalar constraint

Reach the line



#### A single scalar constraint (continued)

matrix equation:

$$\Delta P.\mathbf{n} = \overrightarrow{PP'}.\mathbf{n}$$

$$\begin{pmatrix} \mathbf{a_1} & \mathbf{a_2} \end{pmatrix} \Delta q = \overrightarrow{PP'}.\mathbf{n}$$

$$\begin{pmatrix} a_{1x} & a_{2x} \\ a_{1y} & a_{2y} \end{pmatrix} \begin{pmatrix} \Delta q_1 \\ \Delta q_2 \end{pmatrix} .\mathbf{n} = \overrightarrow{PP'}.\mathbf{n}$$

$$(a_{1x}\Delta q_1 + a_{2x}\Delta q_2)n_x + (a_{1y}\Delta q_1 + a_{2y}\Delta q_2)n_y = \overrightarrow{PP'}.\mathbf{n}$$

$$(a_{1x}n_x + a_{1y}n_y)\Delta q_1 + (a_{2x}n_x + a_{2y}n_y)\Delta q_2 = \overrightarrow{PP'}.\mathbf{n}$$

$$(\mathbf{a_1}.\mathbf{n} \quad \mathbf{a_2}.\mathbf{n}) \begin{pmatrix} \Delta q_1 \\ \Delta q_2 \end{pmatrix} = \overrightarrow{PP'}.\mathbf{n}$$

each constraint can seen as a set of scalar equations

### Singular systems

- Example: coplanar translation axes
- In-plane constraint: infinity of solutions
- Out of the plane: no solution



#### Nonlinear equations

Rotational joints

Several solutions, or no solution at all



#### Linearization - the Jacobian matrix

Starting from the velocity equation, and noticing that  $\frac{dP}{dt} = \frac{dP}{d\mathbf{q}} \frac{d\mathbf{q}}{dt}$ 

$$rac{\delta P}{\delta q_i} = \mathbf{a_i}$$
 (translational dof)  
 $rac{\delta P}{\delta q_i} = \mathbf{a_i} imes \overrightarrow{O_i P}$  (rotational dof)



with n dof:

$$\mathbf{J}_{\rho} = \frac{dP}{d\mathbf{q}} = \left(\begin{array}{cc} \frac{\delta P}{\delta q_1} & \dots & \frac{\delta P}{\delta q_n} \end{array}\right)$$
  
 
$$\Delta P \simeq \mathbf{J}_{\rho} \Delta \mathbf{q}$$

#### Small displacements



$$\left(\begin{array}{cc} \frac{\delta P}{\delta q_1}.\mathbf{n} & \dots & \frac{\delta P}{\delta q_n}.\mathbf{n}\end{array}\right)\Delta \mathbf{q} = b$$

#### Orientation constraints

- Express the rotation from the current orientation to its target and compute the associated axis and angle:  ${}_{n}^{0}\mathbf{R}' = \mathbf{R}_{\mathbf{n},\theta_{n}}{}_{n}^{0}\mathbf{R}$
- express a rotation vector as:  $\Delta r = \theta \mathbf{n}$
- the jacobian matrix is composed of:

$$rac{\delta r}{\delta q_i} = \mathbf{0}$$
 (translational dof)  
 $rac{\delta r}{\delta q_i} = \mathbf{a_i}$  (rotational dof)

- then solve:  $\mathbf{J} \Delta q = \Delta r$
- works for small rotations only

#### Aligning a vector with another

$$\left(\begin{array}{ccc} \frac{\delta r}{\delta q_1}.\mathbf{n} & \dots & \frac{\delta r}{\delta q_n}.\mathbf{n} \\ \frac{\delta r}{\delta q_1}.\mathbf{v} & \dots & \frac{\delta r}{\delta q_v}.\mathbf{v} \end{array}\right) = \left(\begin{array}{c} \theta \\ 0 \end{array}\right)$$



#### Putting all the constraints together

Concatenate the equation systems

$$\left(egin{array}{c} J_0\ dots\ J_n\end{array}
ight)\Delta q=\left(egin{array}{c} c_0\ dots\ c_n\end{array}
ight)$$

### Solve the linear equation system

- square, full-rank matrix: use LU factoring
- more unknowns than equations:

$$\delta \mathbf{q} = \mathbf{J}^+ \mathbf{c}$$
  
with  $\mathbf{J}^+ = \mathbf{J}^T (\mathbf{J} \mathbf{J}^T)^{-1}$ 

gives the smallest solution

more equations than unknowns:

$$\delta \mathbf{q} = (\mathbf{J}^T \mathbf{J})^{-1} \mathbf{J}^T \mathbf{c}$$

gives the closest solution

 when everything has failed, use Singular Value Decomposition (SVD) (chapter 2.6 of Numerical Recipes)

#### Iterative solution of nonlinear equations

```
Newton's algorithm solves a series of linear equation
systems:
```

```
compute constraint vector c
while \|\mathbf{c}\| > \epsilon
compute J
solve J \delta \mathbf{q} = \mathbf{c}
\mathbf{q} \leftarrow \mathbf{q} + \delta \mathbf{q}
compute c
```

# Handling limit values

- Most real-world joints have limit values
- When beyong the limit, project to the limit value and remove the dof from the list: compute constraint vector **c** while  $\|\mathbf{c}\| > \epsilon$ compute J solve  $\mathbf{J} \, \delta \mathbf{q} = \mathbf{c}$  $\mathbf{q} \leftarrow \mathbf{q} + \delta \mathbf{q}$ for each dof i if  $q_i > q_{imax}$  then  $q_i \leftarrow q_{imax}$ remove i from the list of dof compute c

# Exploiting the free space

- When a space of solutions are available (free space), we have room for optimizing quality criteria: equilibrium, comfort, etc.
- Optimize a cost function *e* inside the free space
- project search directions to the free space:

$$\forall \mathbf{z} \ \mathbf{J}(\mathbf{J}^{+}\mathbf{J} - \mathbf{I})\mathbf{z} = \mathbf{J}(\mathbf{J}^{T}(\mathbf{J}\mathbf{J}^{T})^{-1}\mathbf{J} - \mathbf{I})\mathbf{z}$$
  
=  $(\mathbf{J}\mathbf{J}^{T}(\mathbf{J}\mathbf{J}^{T})^{-1}\mathbf{J} - \mathbf{J})\mathbf{z}$   
=  $(\mathbf{J} - \mathbf{J})\mathbf{z}$   
=  $\mathbf{0}$ 

optimization algorithm:

repeat

solve the constraint

do a step toward  $-(\mathbf{J}^+\mathbf{J} - \mathbf{I})\overrightarrow{grad} e$ 

<ロ> <同> <同> < 回> < 三> < 三> 三 三

< □ > < 団 > < 臣 > < 臣 > 王 のへで 34/35